

## Linear dynamical systems with inputs & outputs

- ▶ inputs & outputs: interpretations
- ▶ impulse and step responses
- ▶ examples

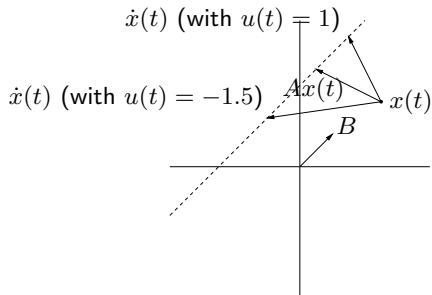
## Inputs & outputs

recall continuous-time time-invariant LDS has form

$$\dot{x} = Ax + Bu, \quad y = Cx + Du$$

- ▶  $Ax$  is called the *drift term* (of  $\dot{x}$ )
- ▶  $Bu$  is called the input term (of  $\dot{x}$ )

picture, with  $B \in \mathbb{R}^{2 \times 1}$ :



## Interpretations

write  $\dot{x} = Ax + b_1u_1 + \dots + b_mu_m$ , where  $B = [b_1 \ \dots \ b_m]$

- ▶ state derivative is sum of autonomous term ( $Ax$ ) and one term per input ( $b_iu_i$ )
- ▶ each input  $u_i$  gives another degree of freedom for  $\dot{x}$  (assuming columns of  $B$  independent)

write  $\dot{x} = Ax + Bu$  as  $\dot{x}_i = \tilde{a}_i^\top x + \tilde{b}_i^\top u$ , where  $\tilde{a}_i^\top, \tilde{b}_i^\top$  are the rows of  $A, B$

- ▶  $i$ th state derivative is linear function of state  $x$  and input  $u$

## Response to input

- ▶ the solution to  $\dot{x} = Ax + Bu$  is

$$x(t) = e^{tA}x(0) + \int_0^t e^{(t-\tau)A}Bu(\tau) d\tau$$

- ▶  $e^{tA}x(0)$  is the unforced or autonomous response
- ▶  $e^{tA}B$  is called the input-to-state impulse response or impulse matrix

## Impulse response

impulse response  $h(t) = Ce^{tA}B + D\delta(t)$

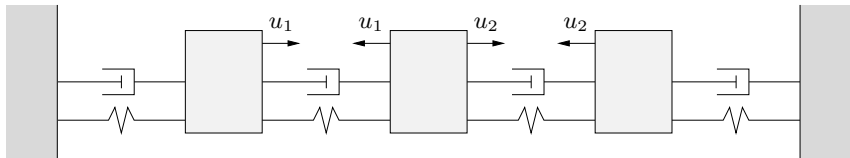
with  $x(0) = 0$ ,  $y = h * u$ , *i.e.*,

$$y_i(t) = \sum_{j=1}^m \int_0^t h_{ij}(t - \tau) u_j(\tau) d\tau$$

### interpretations:

- ▶  $h_{ij}(t)$  is impulse response from  $j$ th input to  $i$ th output
- ▶  $h_{ij}(t)$  gives  $y_i(t)$  when  $u(t) = e_j\delta(t)$
- ▶  $h_{ij}(\tau)$  shows how dependent output  $i$  is, on what input  $j$  was,  $\tau$  seconds ago
- ▶  $i$  indexes output;  $j$  indexes input;  $\tau$  indexes time lag

## Mass-spring example



- ▶ unit masses, springs, dampers
- ▶  $u_1$  is tension between 1st & 2nd masses
- ▶  $u_2$  is tension between 2nd & 3rd masses
- ▶  $y \in \mathbb{R}^3$  is displacement of masses 1,2,3
- ▶  $x = \begin{bmatrix} y \\ \dot{y} \end{bmatrix}$

## Mass-spring example

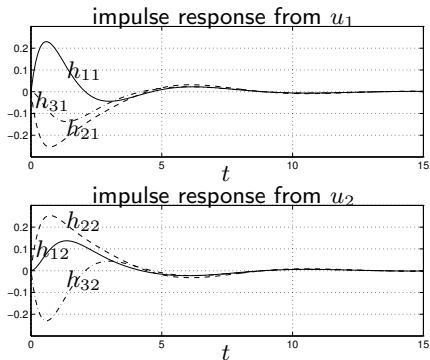
system is:

$$\dot{x} = \begin{bmatrix} 0 & 0 & 0 & 1 & 0 & 0 \\ 0 & 0 & 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 0 & 0 & 1 \\ -2 & 1 & 0 & -2 & 1 & 0 \\ 1 & -2 & 1 & 1 & -2 & 1 \\ 0 & 1 & -2 & 0 & 1 & -2 \end{bmatrix} x + \begin{bmatrix} 0 & 0 \\ 0 & 0 \\ 0 & 0 \\ 1 & 0 \\ -1 & 1 \\ 0 & -1 \end{bmatrix} \begin{bmatrix} u_1 \\ u_2 \end{bmatrix}$$

eigenvalues of  $A$  are

$$-1.71 \pm i0.71, \quad -1.00 \pm i1.00, \quad -0.29 \pm i0.71$$

## Example: Impulse response



roughly speaking:

- ▶ impulse at  $u_1$  affects third mass less than other two
- ▶ impulse at  $u_2$  affects first mass later than other two



## Discretization with piecewise constant inputs

linear system  $\dot{x} = Ax + Bu$ ,  $y = Cx + Du$

suppose  $u_d : \mathbb{Z}_+ \rightarrow \mathbb{R}^m$  is a sequence, and

$$u(t) = u_d(k) \quad \text{for } kh \leq t < (k+1)h, \quad k = 0, 1, \dots$$

define sequences

$$x_d(k) = x(kh), \quad y_d(k) = y(kh), \quad k = 0, 1, \dots$$

- ▶  $h > 0$  is called the *sample interval* (for  $x$  and  $y$ ) or *update interval* (for  $u$ )
- ▶  $u$  is piecewise constant (called *zero-order-hold*)
- ▶  $x_d, y_d$  are sampled versions of  $x, y$

## Discretization with piecewise constant inputs

$$\begin{aligned}x_d(k+1) &= x((k+1)h) \\ &= e^{hA}x(kh) + \int_0^h e^{\tau A}Bu((k+1)h - \tau) d\tau \\ &= e^{hA}x_d(k) + \left(\int_0^h e^{\tau A} d\tau\right) B u_d(k)\end{aligned}$$

$x_d$ ,  $u_d$ , and  $y_d$  satisfy discrete-time LDS equations

$$x_d(k+1) = A_d x_d(k) + B_d u_d(k), \quad y_d(k) = C_d x_d(k) + D_d u_d(k)$$

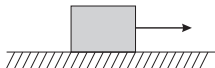
where

$$A_d = e^{hA}, \quad B_d = \left(\int_0^h e^{\tau A} d\tau\right) B, \quad C_d = C, \quad D_d = D$$

called *discretized system*. If  $A$  is invertible, we can express integral as

$$\int_0^h e^{\tau A} d\tau = A^{-1} (e^{hA} - I)$$

## Example: Force on mass



Newton's law gives continuous-time LDS

$$\begin{aligned}\dot{x}(t) &= \begin{bmatrix} 0 & 1 \\ 0 & 0 \end{bmatrix} x(t) + \begin{bmatrix} 0 \\ 1 \end{bmatrix} u(t) \\ y(t) &= \begin{bmatrix} 1 & 0 \end{bmatrix} x(t)\end{aligned}$$

let's compute the discretization

$$\begin{aligned}A_d &= e^{Ah} \\ &= I + Ah + \frac{1}{2}A^2h^2 + \dots \\ &= I + Ah \\ &= \begin{bmatrix} 1 & h \\ 0 & 1 \end{bmatrix}\end{aligned}$$

## Example: Force on mass

$$\begin{aligned} B_d &= \int_0^h e^{As} B ds \\ &= \int_0^h \begin{bmatrix} 1 & s \\ 0 & 1 \end{bmatrix} \begin{bmatrix} 0 \\ 1 \end{bmatrix} ds \\ &= \int_0^h \begin{bmatrix} s \\ 1 \end{bmatrix} ds = \begin{bmatrix} \frac{1}{2}h^2 \\ h \end{bmatrix} \end{aligned}$$

so the discretization is

$$\begin{aligned} x_d(k+1) &= \begin{bmatrix} 1 & h \\ 0 & 1 \end{bmatrix} x_d(k) + \begin{bmatrix} \frac{1}{2}h^2 \\ h \end{bmatrix} u_d(k) \\ y_d(k) &= \begin{bmatrix} 1 & 0 \end{bmatrix} x_d(k) \end{aligned}$$

## Stability of discretization

**stability:** if eigenvalues of  $A$  are  $\lambda_1, \dots, \lambda_n$ , then eigenvalues of  $A_d$  are  $e^{h\lambda_1}, \dots, e^{h\lambda_n}$   
discretization preserves stability properties since

$$\Re\lambda_i < 0 \Leftrightarrow |e^{h\lambda_i}| < 1$$

for  $h > 0$

## Extensions and variations

- ▶ *offsets*: updates for  $u$  and sampling of  $x, y$  are offset in time
- ▶ *multirate*:  $u_i$  updated,  $y_i$  sampled at different intervals  
(usually integer multiples of a common interval  $h$ )

both very common in practice

## Discrete-time systems

discrete-time LDS:

$$x(t+1) = Ax(t) + Bu(t), \quad y(t) = Cx(t) + Du(t)$$

we have:

$$x(1) = Ax(0) + Bu(0),$$

$$\begin{aligned} x(2) &= Ax(1) + Bu(1) \\ &= A^2x(0) + ABu(0) + Bu(1), \end{aligned}$$

and in general, for  $t \in \mathbb{Z}_+$ ,

$$x(t) = A^t x(0) + \sum_{\tau=0}^{t-1} A^{(t-1-\tau)} Bu(\tau)$$

## Discrete-time systems

Solution is

$$x(t) = A^t x(0) + \sum_{\tau=0}^{t-1} A^{(t-1-\tau)} B u(\tau)$$

write this as

$$y(t) = CA^t x(0) + H * u$$

where  $*$  is discrete-time convolution

$$y(t) = CA^t x(0) + \sum_{\tau=0}^t H(t - \tau) u(\tau)$$

and

$$H(t) = \begin{cases} D, & t = 0 \\ CA^{t-1}B, & t > 0 \end{cases}$$

is the impulse response



## Block Toeplitz matrices

we have

$$\begin{bmatrix} y(0) \\ y(1) \\ y(2) \\ \vdots \\ y(t) \end{bmatrix} = \begin{bmatrix} D & & & & \\ CB & D & & & \\ CAB & CB & D & & \\ \vdots & & & \ddots & \\ CA^{t-1}B & CA^{t-2}B & \dots & CB & D \end{bmatrix} \begin{bmatrix} u(0) \\ u(1) \\ u(2) \\ \vdots \\ u(t) \end{bmatrix} + \begin{bmatrix} C \\ CA \\ CA^2 \\ \vdots \\ CA^t \end{bmatrix} x(0)$$

- ▶ this matrix gives the output sequence  $y(0), y(1), \dots$  in terms of the input sequence  $u(0), u(1), \dots$  and the initial state  $x(0)$
- ▶ *block Toeplitz* means blocks are constant along diagonals from top-left to bottom right
- ▶ we can use this to find controllers and estimators

## Example: Point mass

unit point mass, with actuators applying force in directions

$$v_1 = \begin{bmatrix} 1 \\ 0 \end{bmatrix} \quad v_2 = \begin{bmatrix} -0.5 \\ 1 \end{bmatrix} \quad v_3 = \begin{bmatrix} 1 \\ 1 \end{bmatrix}$$

has dynamics

$$x(k+1) = \begin{bmatrix} 1 & h & 0 & 0 \\ 0 & 1 & 0 & 0 \\ 0 & 0 & 1 & h \\ 0 & 0 & 0 & 1 \end{bmatrix} x(k) + \begin{bmatrix} \frac{1}{2}h^2 & 0 \\ h & 0 \\ 0 & \frac{1}{2}h^2 \\ 0 & h \end{bmatrix} [v_1 \quad v_2 \quad v_3] \begin{bmatrix} u_1(k) \\ u_2(k) \\ u_3(k) \end{bmatrix}$$
$$y(k) = \begin{bmatrix} 1 & 0 & 0 & 0 \\ 0 & 0 & 1 & 0 \end{bmatrix} x(k)$$

here

- ▶  $x_1, x_2$  = position, velocity in **x**-direction  
 $x_3, x_4$  = position, velocity in **y**-direction
- ▶  $h$  = sample time; we'll use  $h = 1$ .
- ▶  $u_i(k)$  current applied to actuator  $i$  at time  $k$

## Example: Point mass

we would like to drive it through the positions

$$y(20) = \begin{bmatrix} 5 \\ 3 \end{bmatrix} \quad y(40) = \begin{bmatrix} 10 \\ -1 \end{bmatrix} \quad y(70) = \begin{bmatrix} 4 \\ 1 \end{bmatrix}$$

at the above times

we have

$$y(t) = \sum_{\tau=0}^{t-1} CA^{t-1-\tau} Bu(\tau) + Du(t)$$

this gives the rows of

$$\begin{bmatrix} y(20) \\ y(40) \\ y(70) \end{bmatrix} = A_{\text{act}} \begin{bmatrix} u(0) \\ \vdots \\ u(70) \end{bmatrix}$$

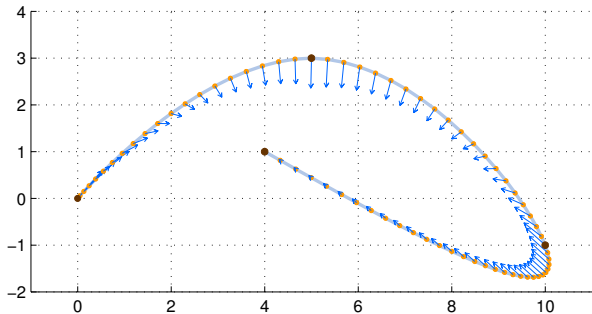
here  $A_{\text{act}}$  is  $6 \times 213$ .

## Example: Point mass

let's find the minimum norm sequence of forces that meets the specifications

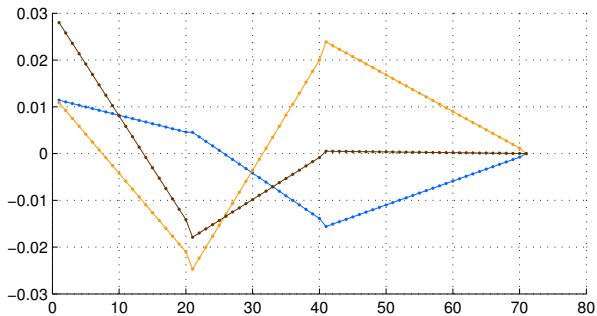
$$\begin{bmatrix} u(0) \\ \vdots \\ u(70) \end{bmatrix} = A_{\text{act}}^{\dagger} \begin{bmatrix} 5 \\ 3 \\ 10 \\ -1 \\ 4 \\ 1 \end{bmatrix}$$

trajectory is



## Example: Point mass

sequence of force inputs is



## Step response

the *step response* or *step matrix* is given by

$$s(t) = \int_0^t h(\tau) d\tau$$

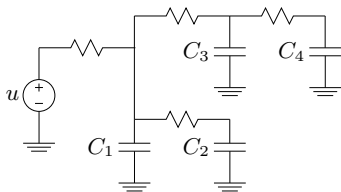
**interpretations:**

- ▶  $s_{ij}(t)$  is step response from  $j$ th input to  $i$ th output
- ▶  $s_{ij}(t)$  gives  $y_i$  when  $u = e_j$  for  $t \geq 0$

for invertible  $A$ , we have

$$s(t) = CA^{-1} \left( e^{tA} - I \right) B + D$$

## Circuit example



- ▶  $u(t) \in \mathbb{R}$  is input (drive) voltage
- ▶  $x_i$  is voltage across  $C_i$
- ▶ output is state:  $y = x$
- ▶ unit resistors, unit capacitors
- ▶ step response matrix shows delay to each node

## Circuit example

system is

$$\dot{x} = \begin{bmatrix} -3 & 1 & 1 & 0 \\ 1 & -1 & 0 & 0 \\ 1 & 0 & -2 & 1 \\ 0 & 0 & 1 & -1 \end{bmatrix} x + \begin{bmatrix} 1 \\ 0 \\ 0 \\ 0 \end{bmatrix} u, \quad y = x$$

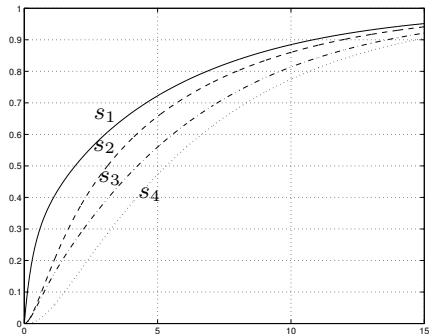
eigenvalues of  $A$  are

$$-0.17, \quad -0.66, \quad -2.21, \quad -3.96$$



## Circuit example

step response matrix  $s(t) \in \mathbb{R}^{4 \times 1}$ :



- ▶ shortest delay to  $x_1$ ; longest delay to  $x_4$
- ▶ delays consistent with slowest (*i.e.*, dominant) eigenvalue  $-0.17$

## DC or static gain matrix

- ▶ DC gain describes system under *static* conditions, *i.e.*,  $x$ ,  $u$ ,  $y$  constant:

$$0 = \dot{x} = Ax + Bu, \quad y = Cx + Du$$

eliminate  $x$  to get  $y = H_0u$  where

$$H_0 = -CA^{-1}B + D$$

- ▶ if system is stable,

$$H_0 = \int_0^{\infty} h(t) dt = \lim_{t \rightarrow \infty} s(t)$$

if  $u(t) \rightarrow u_{\infty} \in \mathbb{R}^m$ , then  $y(t) \rightarrow y_{\infty} \in \mathbb{R}^p$  where  $y_{\infty} = H_0u_{\infty}$

## DC gain matrix

DC gain matrix for spring-mass example:

$$H(0) = \begin{bmatrix} 1/4 & 1/4 \\ -1/2 & 1/2 \\ -1/4 & -1/4 \end{bmatrix}$$

DC gain matrix for circuit example:

$$H(0) = \begin{bmatrix} 1 \\ 1 \\ 1 \\ 1 \end{bmatrix}$$

(do these make sense?)

## Causality

interpretation of

$$x(t) = e^{tA}x(0) + \int_0^t e^{(t-\tau)A}Bu(\tau) d\tau$$

$$y(t) = Ce^{tA}x(0) + \int_0^t Ce^{(t-\tau)A}Bu(\tau) d\tau + Du(t)$$

for  $t \geq 0$ :

*current* state ( $x(t)$ ) and output ( $y(t)$ ) depend on *past* input ( $u(\tau)$  for  $\tau \leq t$ )  
*i.e.*, mapping from input to state and output is *causal* (with fixed *initial* state)

## Idea of state

$x(t)$  is called *state* of system at time  $t$  since:

- ▶ future output depends only on current state and future input
- ▶ future output depends on past input only through current state
- ▶ state summarizes effect of past inputs on future output
- ▶ state is bridge between past inputs and future outputs

## Change of coordinates

start with LDS  $\dot{x} = Ax + Bu$ ,  $y = Cx + Du$

change coordinates in  $\mathbb{R}^n$  to  $\tilde{x}$ , with  $x = T\tilde{x}$

then

$$\dot{\tilde{x}} = T^{-1}\dot{x} = T^{-1}(Ax + Bu) = T^{-1}AT\tilde{x} + T^{-1}Bu$$

hence LDS can be expressed as

$$\dot{\tilde{x}} = \tilde{A}\tilde{x} + \tilde{B}u, \quad y = \tilde{C}\tilde{x} + \tilde{D}u$$

where

$$\tilde{A} = T^{-1}AT, \quad \tilde{B} = T^{-1}B, \quad \tilde{C} = CT, \quad \tilde{D} = D$$

TF is same (since  $u$ ,  $y$  aren't affected):

$$\tilde{C}(sI - \tilde{A})^{-1}\tilde{B} + \tilde{D} = C(sI - A)^{-1}B + D$$

## Standard forms for LDS

can change coordinates to put  $A$  in various forms (diagonal, real modal, Jordan ...)

e.g., to put LDS in *diagonal form*, find  $T$  s.t.

$$T^{-1}AT = \mathbf{diag}(\lambda_1, \dots, \lambda_n)$$

write

$$T^{-1}B = \begin{bmatrix} \tilde{b}_1^\top \\ \vdots \\ \tilde{b}_n^\top \end{bmatrix}, \quad CT = [ \tilde{c}_1 \quad \cdots \quad \tilde{c}_n ]$$